

# TSRT14: Sensor Fusion

## Lecture 5

— Sensor and motion models

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## Lecture 5: sensor and motion models

### Whiteboard:

- Principles and some examples

### Slides:

- Sampling formulas
- Noise models
- Standard motion models
  - Position as integrated velocity, acceleration, . . . , in  $nD$ .
  - Orientation as integrated angular speed in 2D and 3D.
- Odometry

## Lecture 4: summary

- Detection problems as hypothesis tests:

$$H_0 : \mathbf{y} = \mathbf{e},$$

$$H_1 : \mathbf{y} = \bar{x} + \mathbf{e} = \mathbf{h}(x) + \mathbf{e}.$$

- Neyman-Pearson's lemma:  $T(y) = p_{\mathbf{e}}(y - \mathbf{h}(x^0))/p_{\mathbf{e}}(y)$  maximizes  $P_D$  for given  $P_{FA}$  (best ROC curve).
- In general case

$$\bar{T}(y) = 2 \log p_{\mathbf{e}}(y - \mathbf{h}(\hat{x}^{ML})) - 2 \log p_{\mathbf{e}}(y) \sim \chi_{n_x}^2(x^{0,T} \mathcal{I}(x^0)x^0).$$

- Bayes optimal filter

$$p(x_k | y_{1:k}) \propto p_{e_k}(y_k - h(x_k))p(x_k | y_{1:k-1})$$

$$p(x_{k+1} | y_{1:k}) = \int p_{v_k}(x_{k+1} - f(x_k))p(x_k | y_{1:k}) dx_k.$$

- Intuitive work flow of nonlinear filter:

- MU: estimation from  $y_k = h(x_k) + e_k$  and fusion with  $\hat{x}_{k|k-1}$ .

- TU: nonlinear transformation  $z = f(x_k)$  and diffusion from  $x_{k-1} = z_k + v_k$ .

## Modeling and Motion Models

## Chapters 12–14 Overview

- Chapter 12: Principles and methods
  - Principles for deriving discrete time models from continuous time ones
  - Discretized-linearization
  - Linearized-discretization
  - Calibration
- Chapter 13: Motion models
  - Kinematics
  - Rotations
  - Vehicle models
  - Examples
- Chapter 14: Sensor models
  - Techniques
  - Examples

## Modeling

### First problem:

Physics give continuous time model, filters require (linear or nonlinear) discrete time model:

Classification	Nonlinear	Linear
Continuous time	$\dot{x} = a(x, u) + v$ $y = c(x, u) + e$	$\dot{x} = Ax + Bu + v$ $y = Cx + Du + e$
Discrete time	$x_{k+1} = f(x, u) + \bar{v}$ $y = h(x, u) + e$	$x_{k+1} = Fx + Gu + \bar{v}$ $y = Hx + Ju + e$

## Sampling Formulas (1/2)

### Linear time-invariant (LTI) state-space model:

#### Continuous time

$$\dot{x} = Ax + Bu$$

$$y = Cx + Du$$

#### Discrete time

$$x_{k+1} = Fx_k + Gu_k$$

$$y_k = Hx_k + Ju_k$$

$u$  is either input or process noise (then  $J$  denotes cross-correlated noise!).

- **Zero-order hold (ZOH) sampling** assuming the input is piecewise constant:

$$\begin{aligned}
 x(t+T) &= e^{AT}x(t) + \int_0^T e^{A\tau}Bu(t+T-\tau)d\tau \\
 &= \underbrace{e^{AT}}_F x(t) + \underbrace{\int_0^T e^{A\tau}d\tau}_G Bu(t).
 \end{aligned}$$

- **First order hold (FOH) sampling** assuming the input is piecewise linear, is another option.

## Sampling Formulas (2/2)

- **Bilinear transformation (BIL)** assumes band-limited input

$$\frac{2}{T} \frac{\Delta - 1}{\Delta + 1} x(t) \approx \frac{d}{dt} x(t) = Ax + Bu,$$

where  $\Delta$  is the delay operator,  $\Delta x(t) = x(t+T)$ , which yields

$$M = (I_{n_x} - T/2A)^{-1}$$

$$F = M(I_{n_x} + T/2A)$$

$$G = T/2MB$$

$$H = CM$$

$$J = D + HG.$$

## Sampling of Nonlinear Models

There are two options:

- Discretized linearization (general):

1. Linearize:

$$A = \nabla_x a(x, u) \quad B = \nabla_u a(x, u) \quad C = \nabla_x c(x, u) \quad D = \nabla_u c(x, u)$$

2. Discretize (sample):  $F = e^{AT}$ ,  $G = \int_0^T e^{A\tau} d\tau B$ ,  $H = C$ , and  $J = D$

- Linearized discretization (best, if possible!):

1. Discretize (sample nonlinear):

$$x(t+T) = f(x(t), u(t)) = x(t) + \int_t^{t+T} a(x(\tau), u(\tau)) d\tau$$

2. Linearize:  $F = \nabla_x f(x_k, u_k)$  and  $G = \nabla_u f(x_k, u_k)$

## Sampling of State Noise

Different solutions exist, they are all approximations except in the linear case:

- $v_t$  is white noise such that its total influence during one sample interval is  $TQ$  (alternative (12.14d) in the book):

$$\bar{Q}_d = TQ$$

- $v_t$  is a discrete white noise sequence with variance  $TQ$ . That is, we assume that the noise enters immediately after a sample time, so  $x(t+T) = f(x(t) + v(t))$  (alternative (12.14e) in the book):

$$\bar{Q}_e = TGQG^T$$

### Recommendation

In practice simple solutions works well, but *remember to scale with T!*

## Motion Models

Continuous time (physical) and discrete time counterparts

$$\dot{x}(t) = a(x(t), u(t), w(t); \theta)$$

$$x(t+T) = f(x(t), u(t), w(t); \theta, T).$$

- Kinematic models:** Do not attempt to model forces, but are 'Black-box' multi-purpose models.

- Translation kinematics describes position, often based on  $F = ma$ .
- Rotational kinematics describes orientation.
- Rigid body kinematics combines translational and rotational kinematics.
- Constrained kinematics. Coordinated turns (circular path motion).

- Application specific force models**

- Gray-box models** Parameters  $\theta$  must be calibrated (estimated, identified) from data.

## Translational Motion with $n$ Integrators

Translational kinematics models in  $nD$ , where  $p(t)$  denotes:

- Position:  $X(t)$ ,  $(X(t), Y(t))^T$ , or  $(X(t), Y(t), Z(t))^T$
- Rotation:  $\psi(t)$  or  $(\phi(t), \theta(t), \psi(t))^T$

The signal  $w(t)$  is process noise for a pure kinematic model and a motion input signal in position, velocity, and acceleration, respectively, for the case of using sensed motion as an input rather than as a measurement.

State, $x$	Continuous time, $\dot{x}$	Discrete time, $x(t+T)$
$p$	$w$	$x + Tw$
$\begin{pmatrix} p \\ v \end{pmatrix}$	$\begin{pmatrix} 0_n & I_n \\ 0_n & 0_n \end{pmatrix} x + \begin{pmatrix} 0_n \\ I_n \end{pmatrix} w$	$\begin{pmatrix} I_n & TI_n \\ 0_n & I_n \end{pmatrix} x + \begin{pmatrix} T \\ TI_n \end{pmatrix} w$
$\begin{pmatrix} p \\ v \\ a \end{pmatrix}$	$\begin{pmatrix} 0_n & I_n & 0_n \\ 0_n & 0_n & I_n \\ 0_n & 0_n & 0_n \end{pmatrix} x + \begin{pmatrix} 0_n \\ 0_n \\ I_n \end{pmatrix} w$	$\begin{pmatrix} I_n & TI_n & \frac{T^2}{2} I_n \\ 0_n & I_n & TI_n \\ 0_n & 0_n & I_n \end{pmatrix} x + \begin{pmatrix} T \\ \frac{T^2}{2} \\ TI_n \end{pmatrix} w$

## Different Sampled Models of Double Integrator

### Models

$$\dot{x} = Ax + Bu \quad x_{k+1} = Fx_k + Gu_k$$

$$y = Cx + Du \quad y_k = Hx_k + Ju_k$$

$$\text{State: } x = \begin{pmatrix} p(t) \\ v(t) \end{pmatrix}$$

Continuous time	$A = \begin{pmatrix} 0_n & I_n \\ 0_n & 0_n \end{pmatrix}$	$B = \begin{pmatrix} 0_n \\ I_n \end{pmatrix}$	$C = (I_n, 0_n)$	$D = 0_n$
ZOH	$F = \begin{pmatrix} I_n & TI_n \\ 0_n & I_n \end{pmatrix}$	$G = \begin{pmatrix} \frac{T^2}{2} I_n \\ TI_n \end{pmatrix}$	$H = (I_n, 0_n)$	$J = 0_n$
FOH	$F = \begin{pmatrix} I_n & TI_n \\ 0_n & I_n \end{pmatrix}$	$G = \begin{pmatrix} T^2 I_n \\ TI_n \end{pmatrix}$	$H = (I_n, 0_n)$	$J = \frac{T^2}{6} I_n$
BIL	$F = \begin{pmatrix} I_n & TI_n \\ 0 & I \end{pmatrix}$	$G = \begin{pmatrix} \frac{T^2}{2} I_n \\ \frac{T}{2} I \end{pmatrix}$	$H = (I_n, \frac{T}{2} I_n)$	$J = \frac{T^2}{2} I_n$

## Navigation Models

- Navigation models have access to inertial information.
- 2D orientation (course, or yaw rate) much easier than 3D orientation.

## Rotational Kinematics in 2D

The course, or yaw, in 2D can be modeled as integrated white noise

$$\dot{\psi}(t) = w(t),$$

or any higher order of integration. Compare to the tables for translational kinematics with  $p(t) = \psi$  and  $n = 1$ .

## Coordinated Turns in 2D Body Coordinates

Basic motion equations

$$\dot{\psi} = \frac{v_x}{R} = v_x R^{-1},$$

$$a_y = \frac{v_x^2}{R} = v_x^2 R^{-1} = v_x \dot{\psi},$$

$$a_x = \dot{v}_x - v_y \frac{v_x}{R} = \dot{v}_x - v_y v_x R^{-1} = \dot{v}_x - v_y \dot{\psi}.$$

can be combined to a model suitable for the sensor configuration at hand. For instance,

$$x = \begin{pmatrix} \psi \\ R^{-1} \end{pmatrix}, \quad u = v_x, \quad y = R^{-1}$$

$$\dot{x} = f(x, u) + w = \begin{pmatrix} v_x R^{-1} \\ 0 \end{pmatrix} + w$$

is useful when speed is measured, and a vision system delivers a local estimate of (inverse) curve radius.

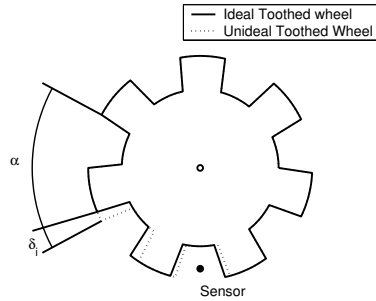
### Automotive Example: wheel speed sensor

Each tooth passing the sensor (electromagnetic or Hall) gives a pulse. The number  $n$  of clock cycles between a number  $m$  of teeth are registered within each sample interval.

$$\omega(t_k) = \frac{2\pi}{N_{\text{cog}}(t_k - t_{k-1})} = \frac{2\pi}{N_{\text{cog}}T_c} \frac{m}{n}$$

**Problems:**

Angle and time quantization. Synchronization. Angle offsets  $\delta$  in sensor teeth.



### Automotive Example: virtual sensors

Longitudinal velocity, yaw rate and slip on left and right driven wheel (front wheel driven assumed) can be computed from wheel angular speeds **if** the radii are known:

$$v_x = \frac{v_3 + v_4}{2} = \frac{\omega_3 r_3 + \omega_4 r_4}{2},$$

$$\dot{\psi} = \frac{\omega_3 r_3 - \omega_4 r_4}{B},$$

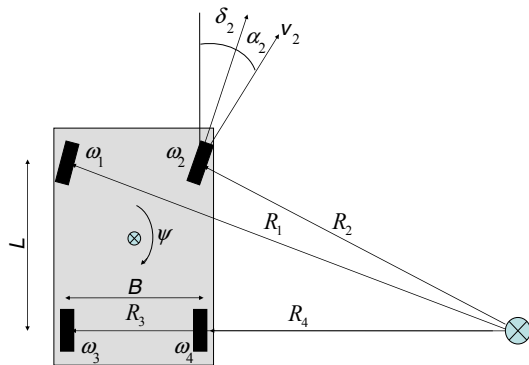
$$s_1 = \frac{\omega_1 r_1}{v_1} - 1, \quad s_2 = \frac{\omega_2 r_2}{v_2} - 1,$$

$$v_1 = v_x \sqrt{\left(1 + \frac{1}{2} R^{-1} B\right)^2 + (R^{-1} L)^2},$$

$$v_2 = v_x \sqrt{\left(1 - \frac{1}{2} R^{-1} B\right)^2 + (R^{-1} L)^2}.$$

### Automotive Example: geometry

The formulas are based on geometry, the relation  $\dot{\psi} = v_x R^{-1}$  and notation below.



### Automotive Example: odometry

Odometry is based on the virtual sensors

$$v_x^m = \frac{\omega_3 r_3 + \omega_4 r_4}{2}$$

$$\dot{\psi}^m = v_x^m \frac{2}{B} \frac{\omega_3 r_3 - \omega_4 r_4}{\omega_3 r_3 + \omega_4 r_4 + 1}$$

and the model

$$\psi_t = \psi_0 + \int_0^t \dot{\psi}_\tau d\tau,$$

$$X_t = X_0 + \int_0^t v_\tau^x \cos(\psi_\tau) d\tau,$$

$$Y_t = Y_0 + \int_0^t v_\tau^x \sin(\psi_\tau) d\tau.$$

to dead-reckon the wheel speeds to a relative position in the global frame.

The position  $(X_t(r_3, r_4), Y_t(r_3, r_4))$  depends on the values of wheel radii  $r_3$  and  $r_4$ .

Further sources of error come from wheel slip in longitudinal and lateral direction. More sensors needed for navigation.

## Rotational Kinematics in 3D

Much more complicated in 3D than 2D! Could be a course in itself.

Coordinate notation for rotations of a body in local coordinate system  $(x, y, z)$  relative to an earth fixed coordinate system:

Motion components	Rotation Euler angle	Angular speed
Longitudinal forward motion $x$	Roll $\phi$	$\omega^x$
Lateral motion $y$	Pitch $\theta$	$\omega^y$
Vertical motion $z$	Yaw $\psi$	$\omega^z$

## Euler Orientation in 3D

An earth fixed vector  $\mathbf{g}$  (for instance the gravitational force) is in the body system oriented as  $Q\mathbf{g}$ , where

$$\begin{aligned}
 Q &= Q_\phi^x Q_\theta^y Q_\psi^z \\
 &= \begin{pmatrix} 1 & 0 & 0 \\ 0 & \cos \phi & \sin \phi \\ 0 & -\sin \phi & \cos \phi \end{pmatrix} \begin{pmatrix} \cos \theta & 0 & -\sin \theta \\ 0 & 1 & 0 \\ \sin \theta & 0 & \cos \theta \end{pmatrix} \begin{pmatrix} \cos \psi & \sin \psi & 0 \\ -\sin \psi & \cos \psi & 0 \\ 0 & 0 & 1 \end{pmatrix} \\
 &= \begin{pmatrix} \cos \theta \cos \psi & \cos \theta \sin \psi & -\sin \theta \\ \sin \phi \sin \theta \cos \psi - \cos \phi \sin \psi & \sin \phi \sin \theta \sin \psi + \cos \phi \cos \psi & \sin \phi \cos \theta \\ \cos \phi \sin \theta \cos \psi + \sin \phi \sin \psi & \cos \phi \sin \theta \sin \psi - \sin \phi \cos \psi & \cos \phi \cos \theta \end{pmatrix}.
 \end{aligned}$$

### Note:

The result depends on the order of rotations  $Q_\phi^x Q_\theta^y Q_\psi^z$ . Here, the  $xyz$  rule is used, but there are other options.

## Euler Rotation in 3D

When the body rotate with  $\omega$ , the Euler angles change according to

$$\begin{pmatrix} \dot{\omega}_x \\ \dot{\omega}_y \\ \dot{\omega}_z \end{pmatrix} = \begin{pmatrix} \dot{\phi} \\ 0 \\ 0 \end{pmatrix} + Q_\phi^x \begin{pmatrix} 0 \\ \dot{\theta} \\ 0 \end{pmatrix} + Q_\phi^x Q_\theta^y \begin{pmatrix} 0 \\ 0 \\ \dot{\psi} \end{pmatrix}.$$

The dynamic equation for Euler angles can be derived from this as

$$\begin{pmatrix} \dot{\phi} \\ \dot{\psi} \\ \dot{\theta} \end{pmatrix} = \begin{pmatrix} 1 & \sin(\phi) \tan(\theta) & \cos(\phi) \tan(\theta) \\ 0 & \cos(\phi) & -\sin(\phi) \\ 0 & \sin(\phi) \sec(\theta) & \cos(\phi) \sec(\theta) \end{pmatrix} \begin{pmatrix} \omega_x \\ \omega_y \\ \omega_z \end{pmatrix}.$$

*Singularities* (gimbal lock) when  $\theta = \pm \frac{\pi}{2}$ , can cause numeric divergence!

## Unit Quaternions

- Vector representation:  $q = (q^0, q^1, q^2, q^3)^T$ .
- Norm constraint of unit quaternion:  $\|q\| = q^T q = 1$ .
- The quaternion can be interpreted as an axis angle:

$$q = \begin{pmatrix} \cos(\frac{1}{2}\alpha) \\ \sin(\frac{1}{2}\alpha)\hat{v} \end{pmatrix},$$

where  $q$  represents a rotation with  $\alpha$  around the axis defined by  $\hat{v}$ ,  $\|\hat{v}\| = 1$ .

### Pros and Cons

- + No singularity.
- + No  $2\pi$  ambiguity.
- More complex and non-intuitive algebra.
- The norm must be maintained; this can be handled. by projection or as a virtual measurement with small noise.

## Quaternion Orientation in 3D

The orientation of the vector  $\mathbf{g}$  in body system is  $Q\mathbf{g}$ , where

$$Q = \begin{pmatrix} q_0^2 + q_1^2 - q_2^2 - q_3^2 & 2q_1q_2 - 2q_0q_3 & 2q_0q_2 + 2q_1q_3 \\ 2q_0q_3 + 2q_1q_2 & q_0^2 - q_1^2 + q_2^2 - q_3^2 & -2q_0q_1 + 2q_2q_3 \\ -2q_0q_2 + 2q_1q_3 & 2q_2q_3 + 2q_0q_1 & q_0^2 - q_1^2 - q_2^2 + q_3^2 \end{pmatrix} \\ = \begin{pmatrix} 2q_0^2 + 2q_1^2 - 1 & 2q_1q_2 - 2q_0q_3 & 2q_1q_3 + 2q_0q_2 \\ 2q_1q_2 + 2q_0q_3 & 2q_0^2 + 2q_2^2 - 1 & 2q_2q_3 - 2q_0q_1 \\ 2q_1q_3 - 2q_0q_2 & 2q_2q_3 + 2q_0q_1 & 2q_0^2 + 2q_3^2 - 1 \end{pmatrix}.$$

## Quaternion Rotation in 3D

Rotation with  $\omega$  gives a dynamic equation for  $q$  which can be written in two equivalent forms:

$$\dot{q} = \frac{1}{2}S(\omega)q = \frac{1}{2}\bar{S}(q)\omega,$$

where

$$S(\omega) = \begin{pmatrix} 0 & -\omega_x & -\omega_y & -\omega_z \\ \omega_x & 0 & \omega_z & -\omega_y \\ \omega_y & -\omega_z & 0 & \omega_x \\ \omega_z & \omega_y & -\omega_x & 0 \end{pmatrix}, \quad \bar{S}(q) = \begin{pmatrix} -q_1 & -q_2 & -q_3 \\ q_0 & -q_3 & q_2 \\ q_3 & q_0 & -q_1 \\ -q_2 & q_1 & q_0 \end{pmatrix}.$$

## Sampled Form of Quaternion Dynamics

The ZOH sampling formula

$$q(t+T) = e^{\frac{1}{2}S(\omega(t))T}q(t)$$

actually has a closed form solution

$$q(t+T) = \left( \cos\left(\frac{T}{2}\|\omega(t)\|\right)I_4 + \frac{\text{sinc}\left(\frac{T}{2}\|\omega(t)\|\right)}{\frac{T}{2}\|\omega(t)\|} \overbrace{\sin\left(\frac{T}{2}\|\omega(t)\|\right)}^{\text{sinc}\left(\frac{T}{2}\|\omega(t)\|\right)} S(\omega(t)) \right) q(t) \\ \approx \left( I_4 + \frac{T}{2}S(\omega(t)) \right) q(t).$$

The approximation coincides with Euler forward sampling approximation, and has to be used in more complex models where, e.g.,  $\omega$  is part of the state vector.

## Double Integrated Quaternion

$$\begin{pmatrix} \dot{q}(t) \\ \dot{\omega}(t) \end{pmatrix} = \begin{pmatrix} \frac{1}{2}S(\omega(t))q(t) \\ w(t) \end{pmatrix}.$$

There is no known closed form discretized model. However, the approximate form can be discretized using the chain rule to

$$\begin{pmatrix} q(t+T) \\ \omega(t+T) \end{pmatrix} \approx \underbrace{\begin{pmatrix} I_4 \frac{T}{2} S(\omega(t)) & \frac{T}{2} \bar{S}(q(t)) \\ 0_{3 \times 4} & I_3 \end{pmatrix}}_{F(t)} \begin{pmatrix} q(t) \\ \omega(t) \end{pmatrix} \\ + \underbrace{\begin{pmatrix} \frac{T^3}{4} S(\omega(t)) I_4 \\ T I_3 \end{pmatrix}}_{G(t)} v(t).$$

## Rigid Body Kinematics

A “multi-purpose” model for all kind of navigation problems in 3D (22 states)

$$\begin{pmatrix} \dot{p} \\ \dot{v} \\ \dot{a} \\ \dot{q} \\ \dot{\omega} \\ \dot{b}^{\text{acc}} \\ \dot{b}^{\text{gyro}} \end{pmatrix} = \begin{pmatrix} 0 & I & 0 & 0 & 0 \\ 0 & 0 & I & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & -\frac{1}{2}S(\omega) & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \end{pmatrix} \begin{pmatrix} p \\ v \\ a \\ q \\ \omega \\ b^{\text{acc}} \\ b^{\text{gyro}} \end{pmatrix} + \begin{pmatrix} 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \\ 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{pmatrix} \begin{pmatrix} v^a \\ v^\omega \\ v^{\text{acc}} \\ v^{\text{gyro}} \end{pmatrix}.$$

Bias states for the accelerometer and gyroscope have been added as well.

## Sensor Model for Kinematic Model

Inertial sensors (gyroscope, accelerometer, magnetometer) are used as sensors.

$$\begin{aligned} y_t^{\text{acc}} &= R(q_t)(a_t - \mathbf{g}) + b_t^{\text{acc}} + e_t^{\text{acc}}, & e_t^{\text{acc}} &\sim \mathcal{N}(0, R_t^{\text{acc}}), \\ y_t^{\text{mag}} &= R(q_t)\mathbf{m} + b_t^{\text{mag}} + e_t^{\text{mag}}, & e_t^{\text{mag}} &\sim \mathcal{N}(0, R_t^{\text{mag}}), \\ y_t^{\text{gyro}} &= \omega_t + b_t^{\text{gyro}} + e_t^{\text{gyro}}, & e_t^{\text{gyro}} &\sim \mathcal{N}(0, R_t^{\text{gyro}}). \end{aligned}$$

Bias observable, but special calibration routines are recommended:

**Stand-still detection:** When  $\|y_t^{\text{acc}}\| \approx \mathbf{g}$  and/or  $\|y_t^{\text{gyro}}\| \approx 0$ , the gyro and acc bias is readily read off. Can decrease drift in dead-reckoning from cubic to linear.

**Ellipse fitting:** When “waving the sensor” over short time intervals, the gyro can be integrated to give accurate orientation, and acc and magnetometer measurements can be transformed to a sphere from an ellipse.

## Tracking Models

- Navigation models have access to inertial information, tracking models have not.
- Orientation mainly the direction of the velocity vector.
- Course (yaw rate) critical parameter.
- Less differences between the 2D and 3D cases.

## Coordinated Turns in 2D World Coordinates

Cartesian velocity	Polar velocity
$\dot{X} = v^X$	$\dot{X} = v \cos(h)$
$\dot{Y} = v^Y$	$\dot{Y} = v \sin(h)$
$\dot{v}^X = -\omega v^Y$	$\dot{v} = 0$
$\dot{v}^Y = \omega v^X$	$\dot{h} = \omega$
$\dot{\omega} = 0$	$\dot{\omega} = 0$
$A = \begin{pmatrix} 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & -\omega & -v^Y \\ 0 & 0 & \omega & 0 & v^X \\ 0 & 0 & 0 & 0 & 0 \end{pmatrix}$	$A = \begin{pmatrix} 0 & 0 & \cos(h) & -v \sin(h) & 0 \\ 0 & 0 & \sin(h) & v \cos(h) & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 \\ 0 & 0 & 0 & 0 & 0 \end{pmatrix}$
$X_{t+T} = X + \frac{v^X}{\omega} \sin(\omega T) - \frac{v^Y}{\omega} (1 - \cos(\omega T))$	$X_{t+T} = X + \frac{2v}{\omega} \sin(\frac{\omega T}{2}) \cos(h + \frac{\omega T}{2})$
$Y_{t+T} = Y + \frac{v^X}{\omega} (1 - \cos(\omega T)) + \frac{v^Y}{\omega} \sin(\omega T)$	$Y_{t+T} = Y - \frac{2v}{\omega} \sin(\frac{\omega T}{2}) \sin(h + \frac{\omega T}{2})$
$v_{t+T}^X = v^X \cos(\omega T) - v^Y \sin(\omega T)$	$v_{t+T} = v$
$v_{t+T}^Y = v^X \sin(\omega T) + v^Y \cos(\omega T)$	$h_{t+T} = h + \omega T$
$\omega_{t+T} = \omega$	$\omega_{t+T} = \omega$



# Summary

## Summary Lecture 5

- Standard models in global coordinates:

- Translation  $p_t^{(m)} = w_t^p$ .
- 2D orientation for heading  $h_t^{(m)} = w_t^h$ .
- Coordinated turn model

$$\begin{aligned}\dot{X} &= v^X \\ \dot{v}^X &= -\omega v^Y \\ \dot{\omega} &= 0.\end{aligned}$$

$$\begin{aligned}\dot{Y} &= v^Y \\ \dot{v}^Y &= \omega v^X\end{aligned}$$

- Standard models in local coordinates  $(x, y, \psi)$ :

- Odometry and dead reckoning for  $(x, y, \psi)$

$$\begin{aligned}X_t &= X_0 + \int_0^t v_\tau^x \cos(\psi_\tau) d\tau \\ \psi_t &= \psi_0 + \int_0^t \dot{\psi}_\tau d\tau.\end{aligned}$$

$$Y_t = Y_0 + \int_0^t v_\tau^y \sin(\psi_\tau) d\tau$$

- Force models for  $(\dot{\psi}, a_y, a_x, \dots)$ .
- 3D orientation  $\dot{q} = \frac{1}{2}S(\omega)q = \frac{1}{2}\bar{S}(q)\omega$ .